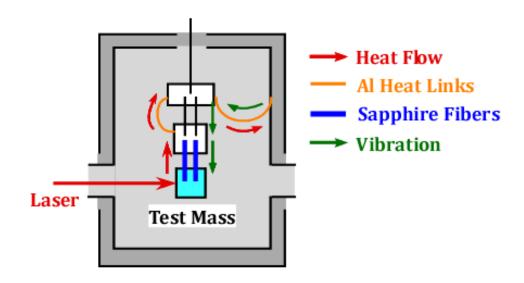




About this Talk



- * Estimate seismic noise introduced from heat links.
- * Discuss how to achieve cooling and seismic isolation at the same time.







Seismic Noise from Heat Links

KAGRA

Basic Requirement for KAGRA Test Mass Suspension

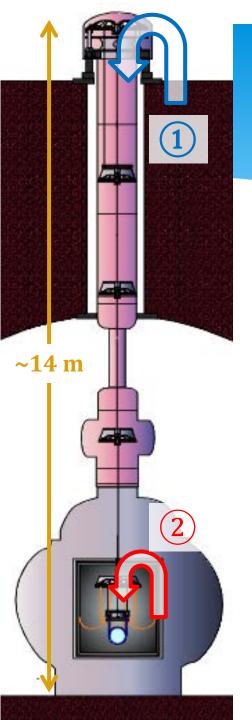


* Attenuate seismic noise

Test Mass Displacement $< 3 \times 10^{-20} \text{ m}/\sqrt{\text{Hz}}$ @ 10 Hz

* Cool down test masses

Mirror Temp. ~ 20 K



Seismic Attenuation System for KAGRA Test Mass



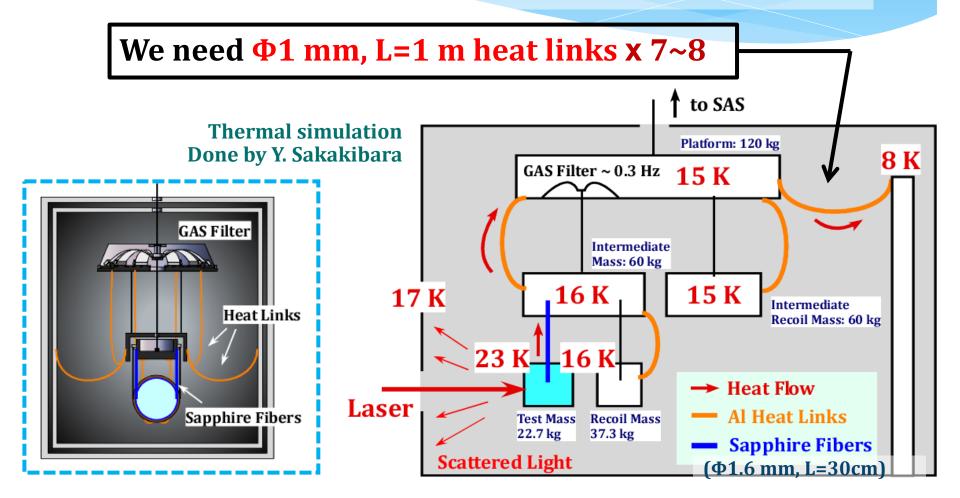
- * Seismic vibration transmits to the mirror in **two different paths**
 - 1. From the top through the attenuation chain
 - 2. From the wall of the cryostat through heat links



Cryogenics



* Heat transferred via pure aluminum heat links.

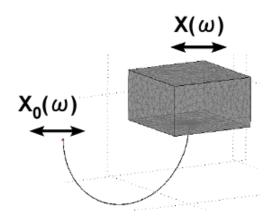




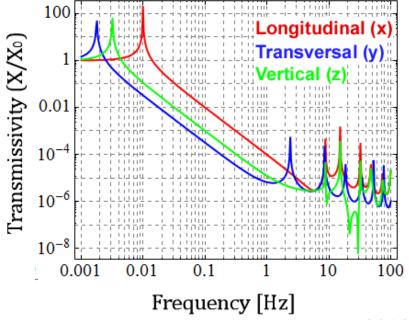
Mechanical Property of Heat Links



* A heat link works as a soft spring (f₀~10 mHz) with **violin modes** above ~1 Hz



FEM Simulation Done by Y. Aso, (JGW-G1000108)



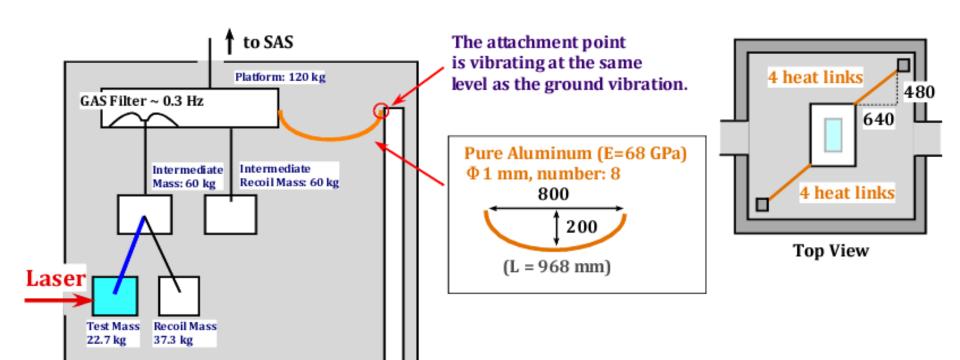


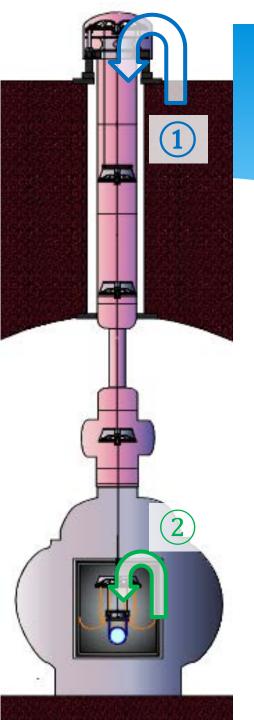
KAGRA Estimation of Seismic Noise via Heat Links



* Simulation Tool: 3-D rigid-body model simulation

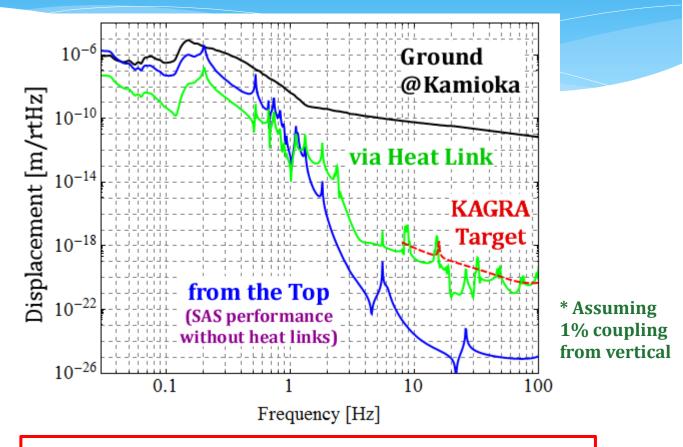
T. Sekiguchi, Master Thesis (JGW-P1200770)





Calculation Result





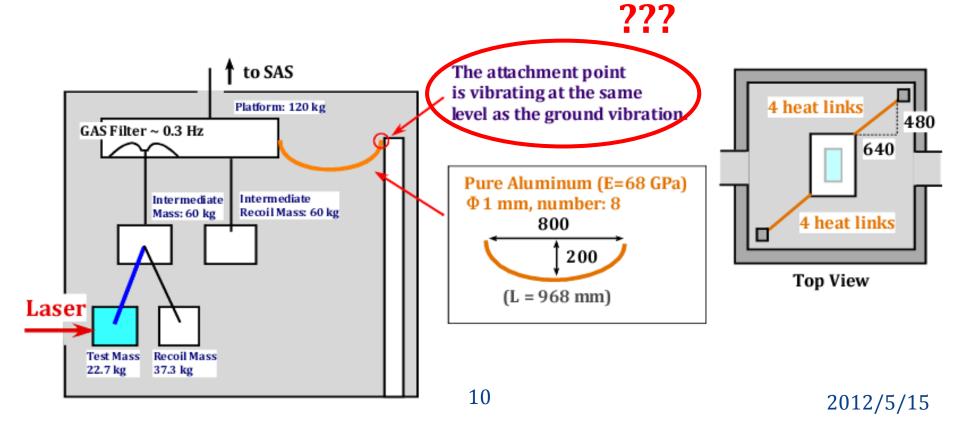
Polluting detector sensitivity above 10 Hz!!



To Make Matters Worse ...



* There is no guarantee that the wall inside the cryostat is vibrating at the same level as the ground.

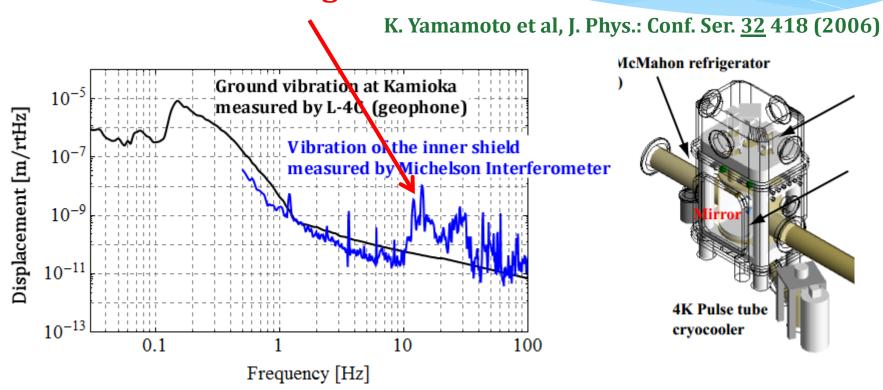


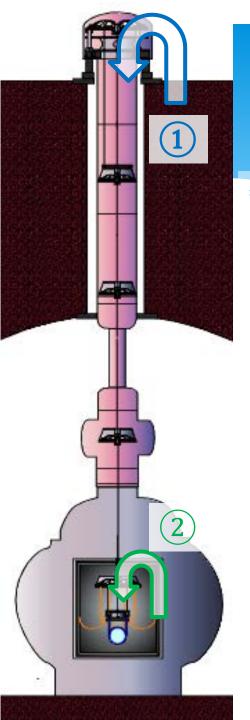


Vibration Inside the Cryostat in CLIO



10-100 times larger !!

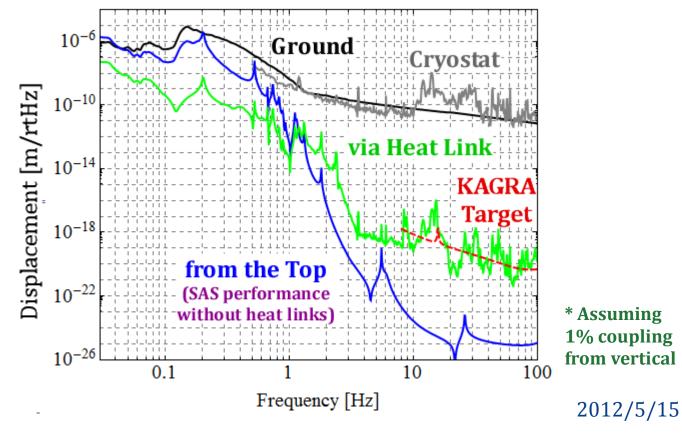




In Worse Case



* If the attachment point of heat links is vibrating at the same level as cryostat vibration in CLIO...

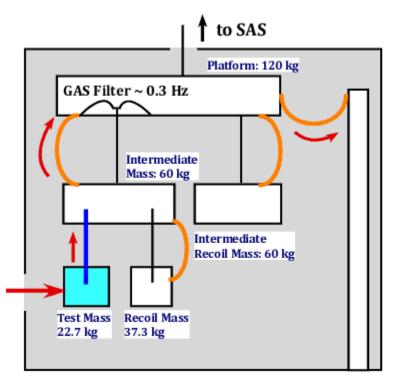


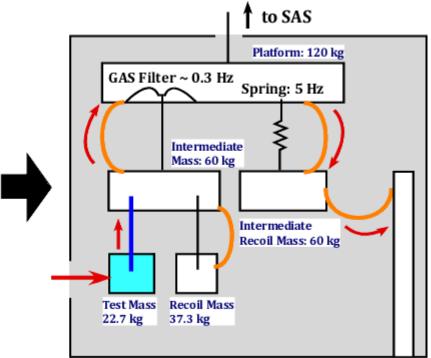


Improved Design



* Add one more "cushion" between the cryostat and mirror



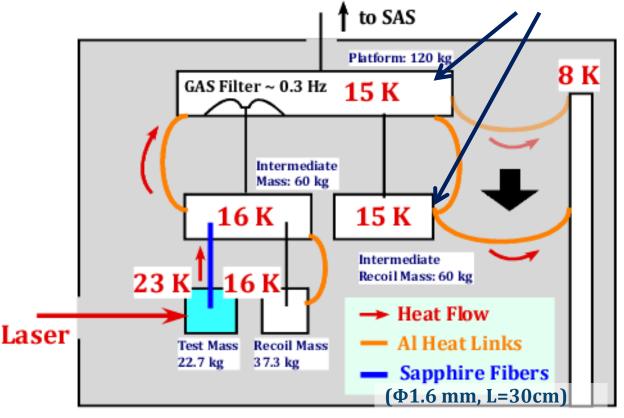




Consideration on Cooling



These two masses are thermally well connected by heat links*

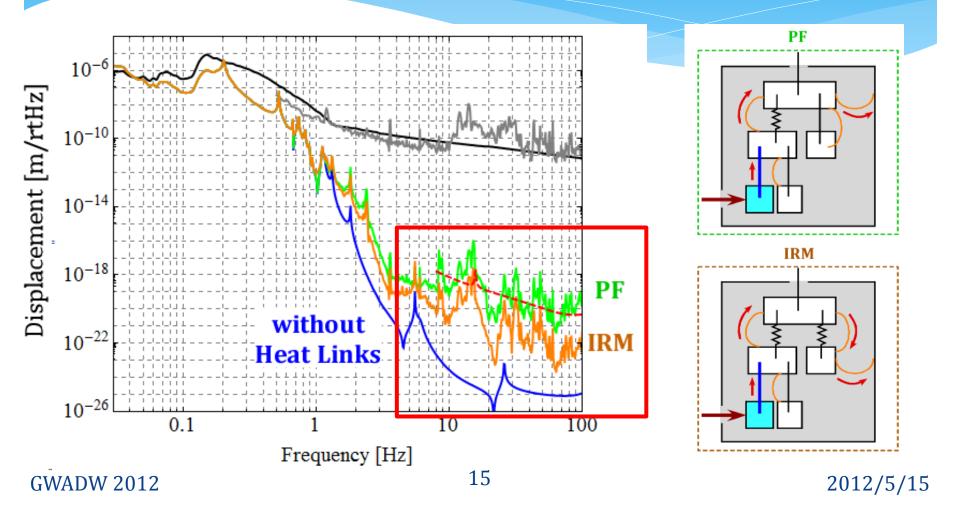


* Pure aluminum Ф 3 mm, L=63 cm, Number: 5



After Improving Wiring

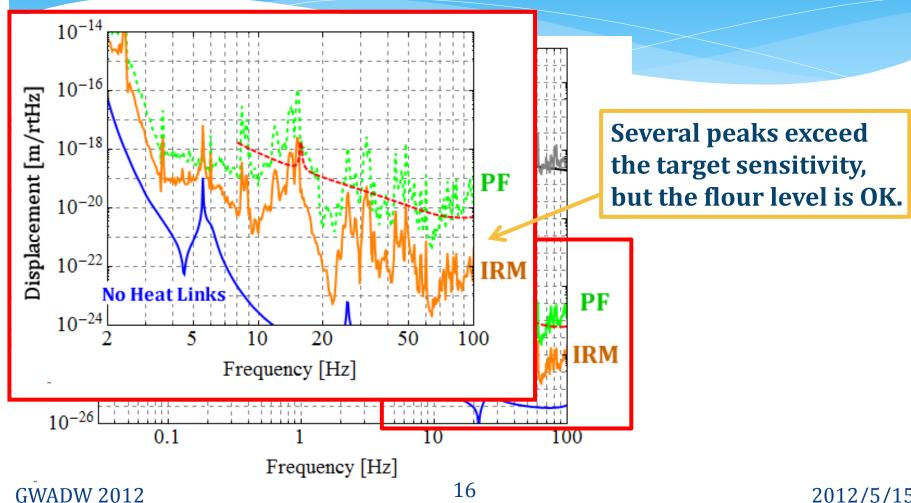






After Improving Wiring



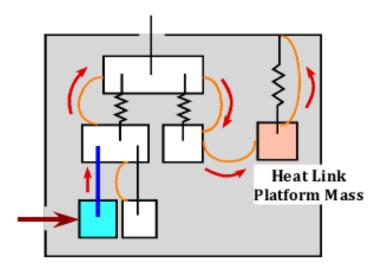




Possible Ideas of Further Improvement



- * Suppress the cryostat vibration passively/actively.
- * Put additional filters between suspension and cryostat.



* Add vertical springs for test mass suspension.





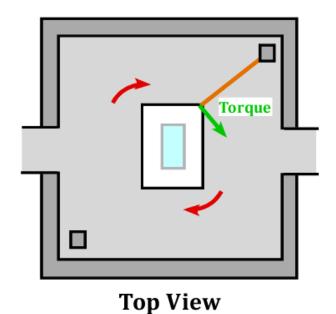
Another Consideration



Effect on Angular Motion



- * SAS is **very soft** in yaw motion (~ 10 mHz).
- * Low frequency yaw motion can be easily excited.



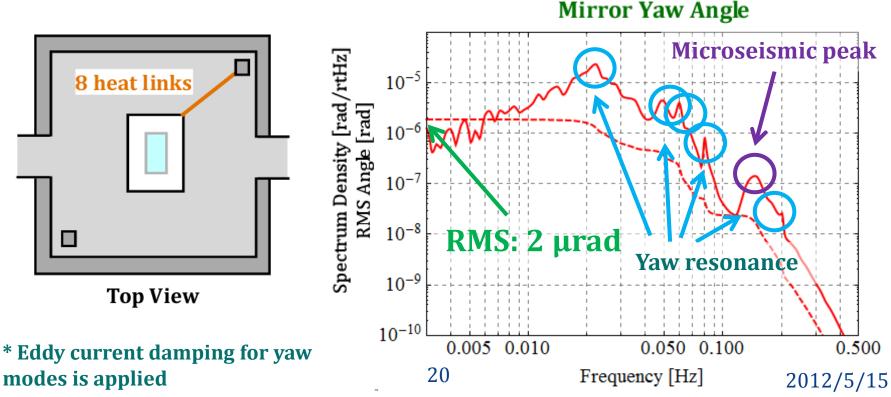
Single wire suspension



Effect on Angular Motion



* If one employs asymmetric wiring of heat links...



T. Sekiguchi, Master Thesis (JGW-P1200770)

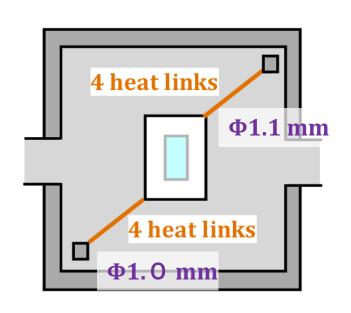


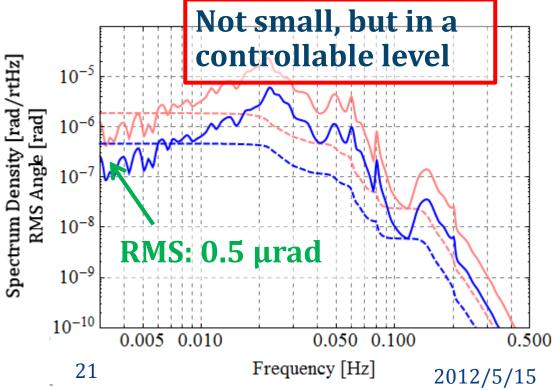
Symmetric Configuration



- * Symmetric wiring does not subject any torque.
- * If you admit 10% thickness difference in two connections

→ Blue Curve









Summary



Summary



- * Careful wiring of heat links is required, in order to mitigate the seismic noise introduced from them.
- * Further isolation, or improvement of the suspension design may be necessary.
- * Yaw excitation by heat links would be **not** so huge (be in a controllable level).

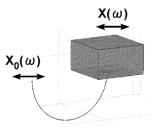


Future Works

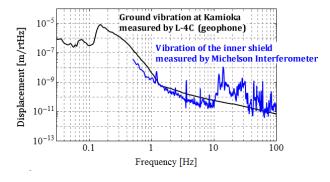


* Transfer function measurement of heat links.

(How to ??)



- * Vibration measurement inside the cryostat.
 - * L. Naticchioni and D. Chen will start this autumn







The END









Appendices



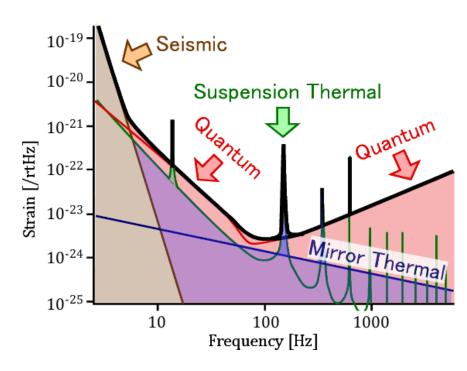
Requirement for KAGRA Test-Mass Suspensions (1)



* Seismic noise should be much lower (at least 10 times smaller) than other noises at the detector observation band (> 10 Hz).

Seismic Noise Requirement: $< 3 \times 10^{-20} \text{ m}/\sqrt{\text{Hz}} @ 10 \text{ Hz}$ And rolls off steeper than f³

KAGRA Design Sensitivity

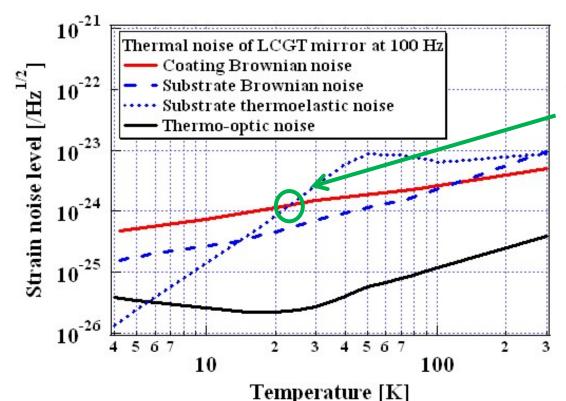




Requirement for KAGRA Test-Mass Suspensions (2)



* Mirror temperature should be as low as 20 K to suppress thermal noise.

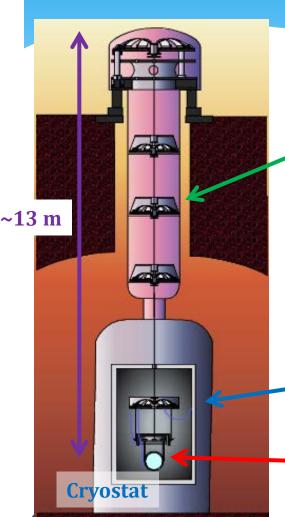


Substrate thermoelastic noise ($\propto T^{2.5}$) gets lower than coating Brownian noise($\propto T^{0.5}$) at < 23 K



Seismic Attenuation System (SAS) for KAGRA





* 7-stage pendulum + 5-stages vertical spring (horizontal attenuation) (vertical attenuation)

Metal cantilever springs



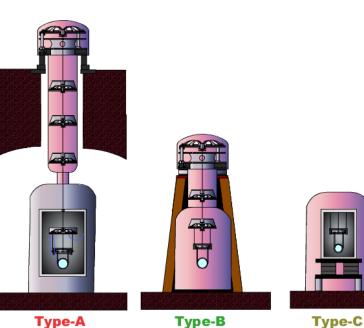
Geometric Anti-Spring (GAS) Filter

 $f_0 \sim 0.3 \text{ Hz}$

Last 3 stages are cooled at **cryogenic** temperature (<20 K) to suppress **thermal noise**

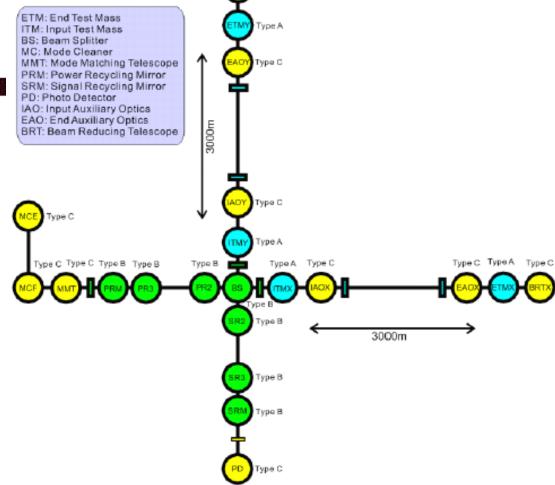
Mirror (Test Mass)

* Beam splitter and other optics are suspended by smaller vibration isolation systems

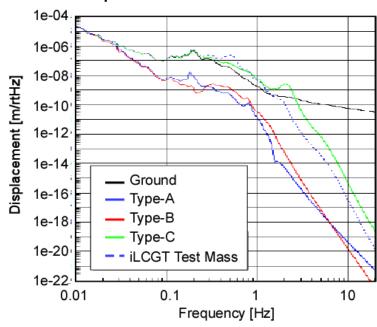


Vibration Isolation System Disposition





Expected Performance





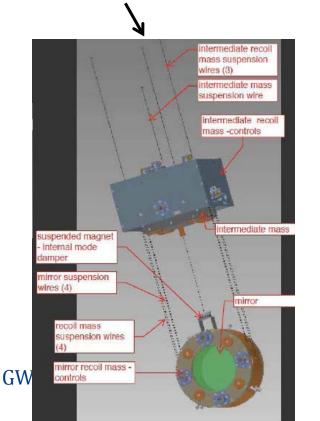
SAS Status

Prototype Experiment:

Standard filter (GASF): Performance was measured @NIKHEF Pre-isolator (IP&GASF): Now Measuring @ Kashiwa

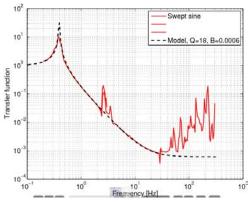
Design:

Type-B Payload: Now Designing









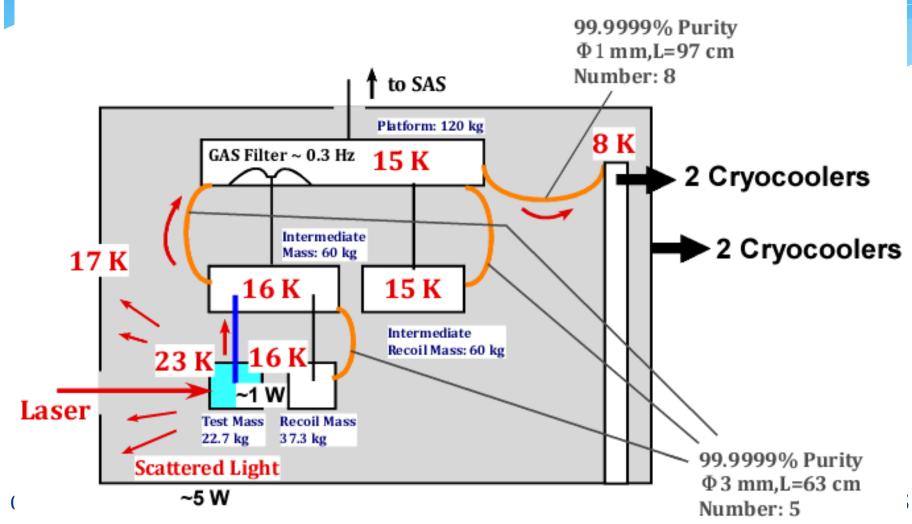
Measured Transfer Function (Feb. 16th, 2011)

2012/5/15



Cryogenics

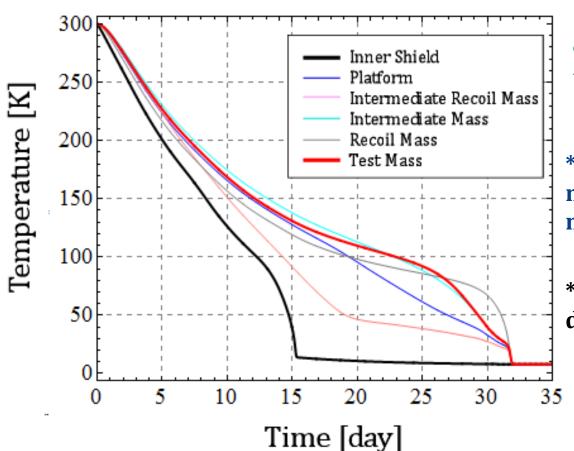






Initial Cooling Time





Thermal simulation with Y. Sakakibara's Method

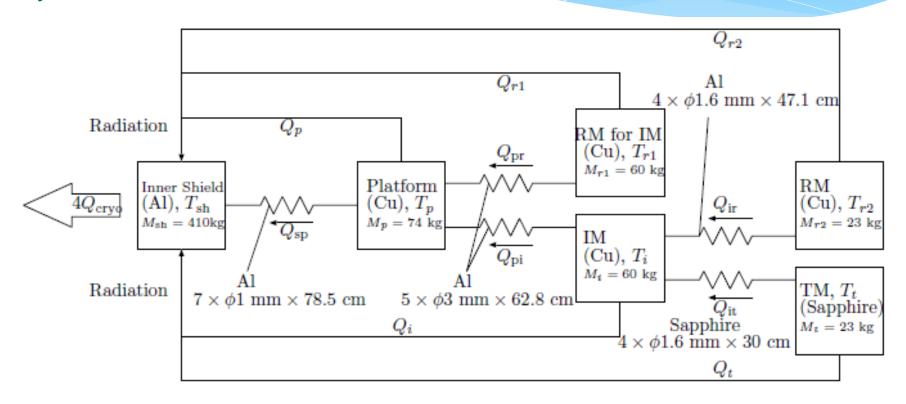
- * The inner shield and the masses except for the test mass are DLC-coated (ϵ =0.41).
- ** Radiation cooling is dominant before 15th day.



Initial Cooling Time Calculation Diagram



By Y. Sakakibara





Heat Load



* Absorption in mirror

Coating: 0.4 W (1ppm)

Substrate: 0.6 W (50 ppm/cm)

Total: 1.0 W

* Inner shield

Radiation from 80 K: 1.3 W

Conductance: 0.8 W

Scattered Light: **5 W** (10 ppm)

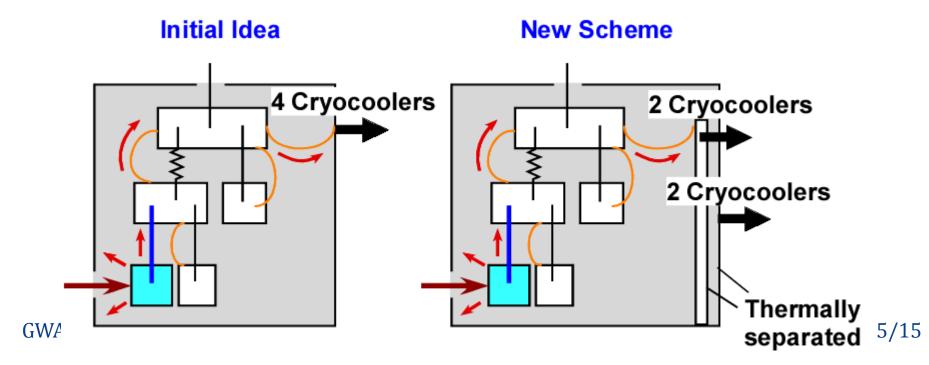
Total: 7.1 W



Heat Extraction Scheme



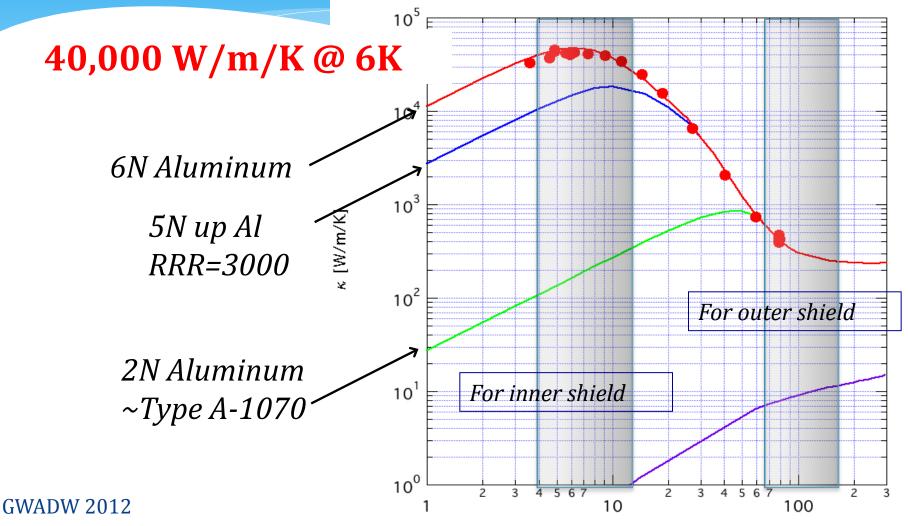
* In the new heat extraction scheme, **mirror temperature** would not be raised, even if large **scattered light** attacks the shield.





Thermal Conductivity of High Pure Aluminum





T [K]



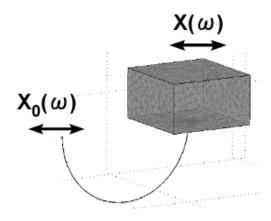
Heat Link TF Calculation

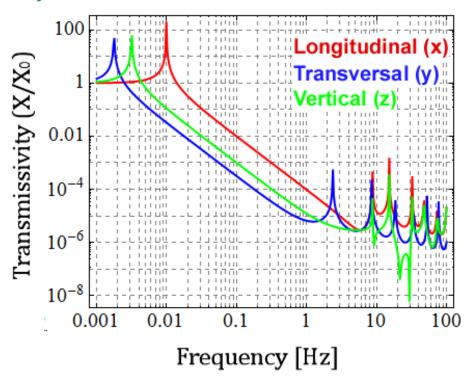




FEM Simulation Done by Y. Aso, (JGW-G1000108)

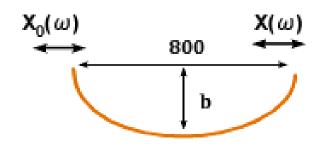
- * Pure aluminum (E=68 GPa), Φ1 mm
- * Half-ellipsoid (a=400 mm, b=200 mm)
- * Loss angle: 10⁻⁴

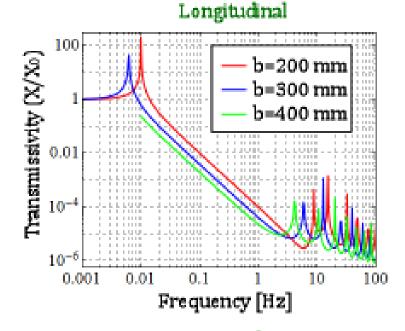


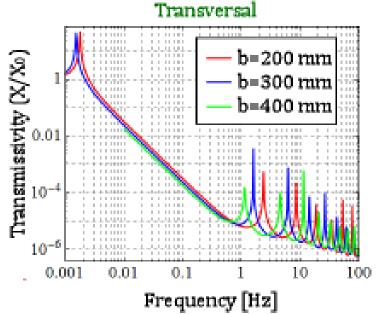


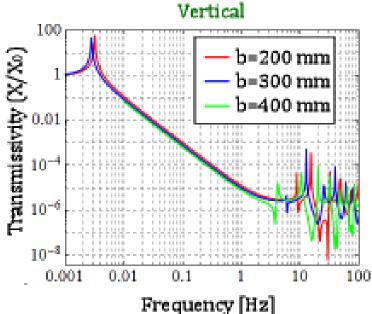


Heat Link Transfer Functions



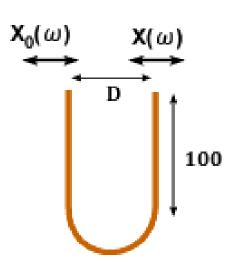




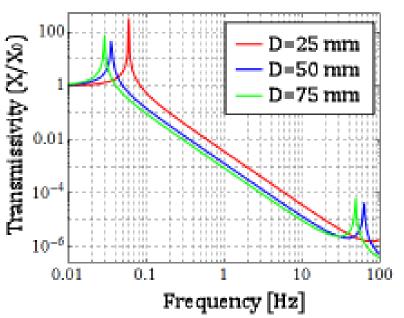




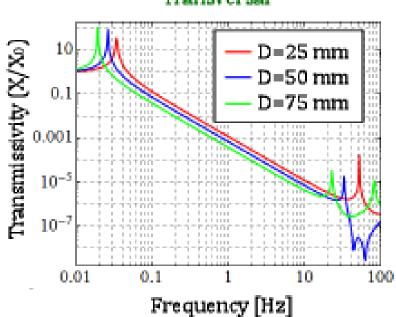
Heat Link TFs



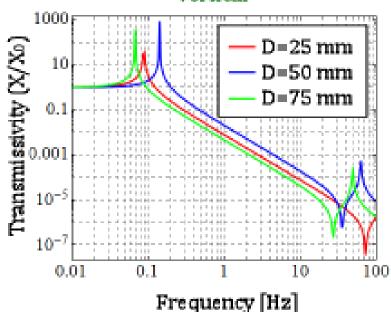
Longitudinal







Vertical



GWADW 2

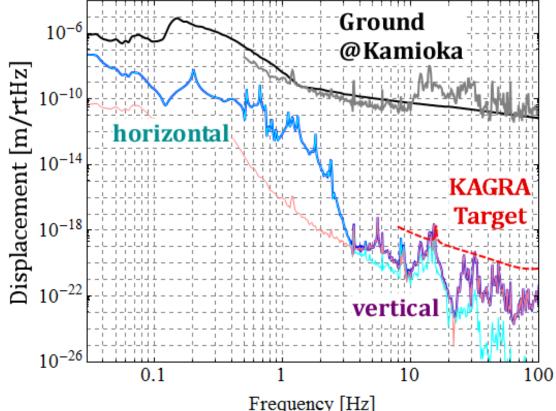


Seismic Noise via Heat Links



Couplings from the **vertical** motion is dominant above 3 Hz.

Seismic Noise via Heat Links



* Assuming 1% coupling from vertical

GWADW 20

2012/5/15



Heat Links with Half Diameters

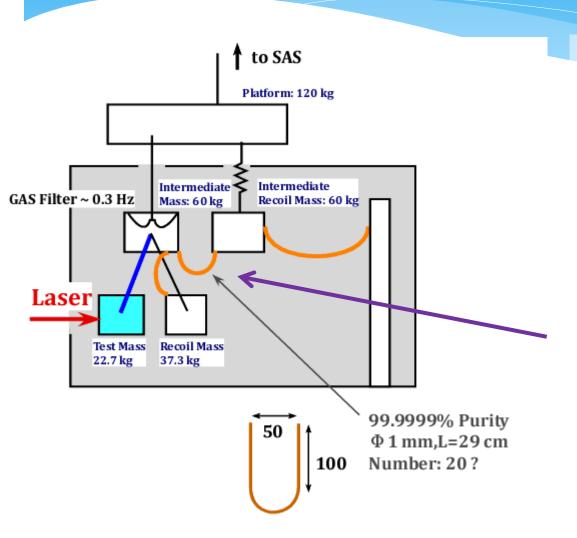


- * How about decreasing the fiber thickness, from $\Phi 1.0 \text{ mm}$ to $\Phi 0.5 \text{ mm}$?
 - * Heat conductivity per link: x 1/4
 - * Necessary number of links: x 4
 - * Spring constant per link: x 1/16
 - * Total stiffness: 1/4
 - * Heat link total mass: Same



Hot Platform Design





Push heavy Platform to the **room temperature** part.

Decrease the **initial cooling time.**

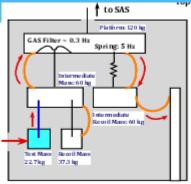
Another vibration shortcut occurs between IM and IRM



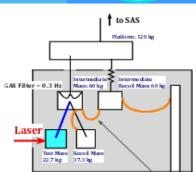
Hot Platform Initial Cooling Time

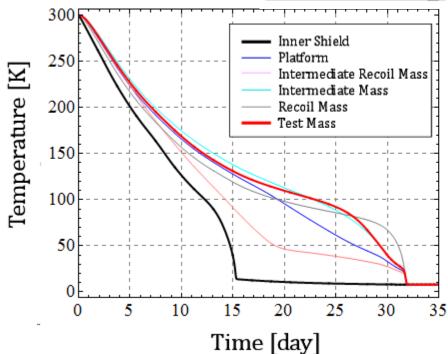


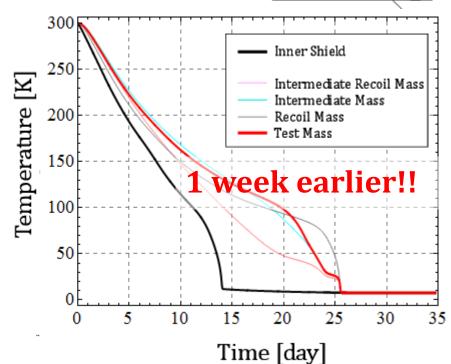
Current Design



Hot Platform

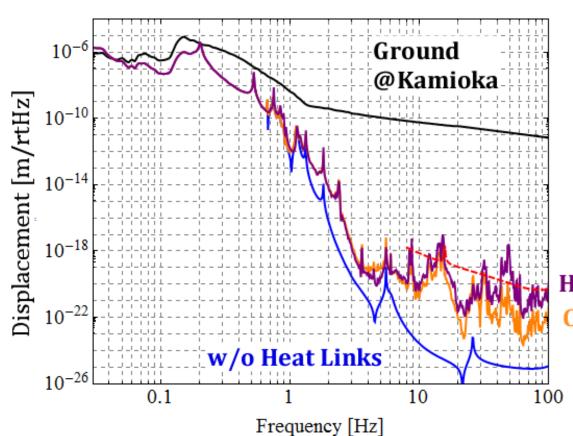


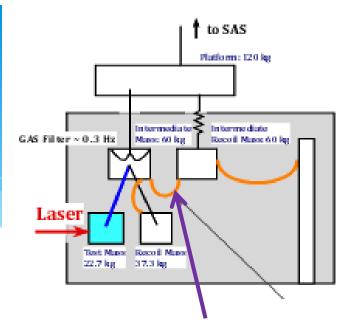






Hot Platform Seismic Noise





Due to the **vibration shortcut**, seismic noise
gets larger **above 10 Hz**in ~1 order of magnitude

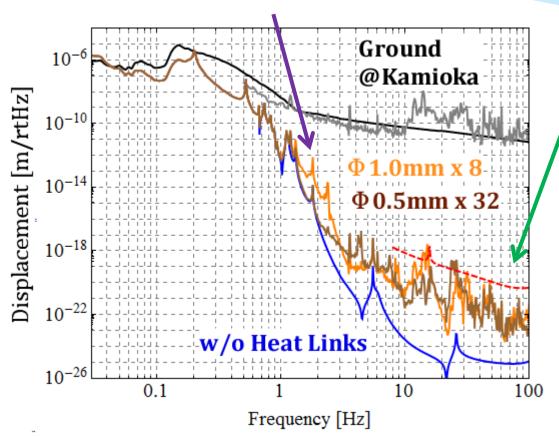
Hot Platform Current Design



Heat Links with Half Diameters



* Only better at 1-3 Hz, not so good above 10 Hz



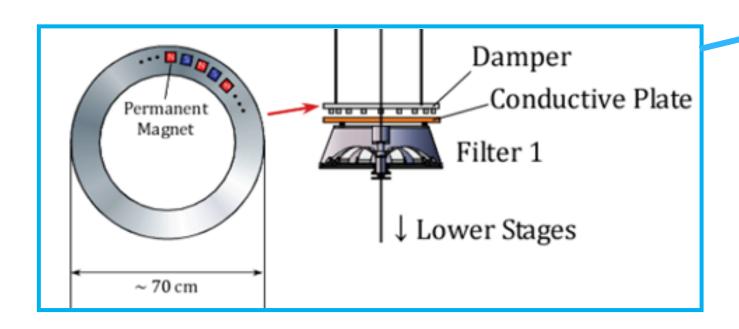
* Assuming 1% coupling from vertical

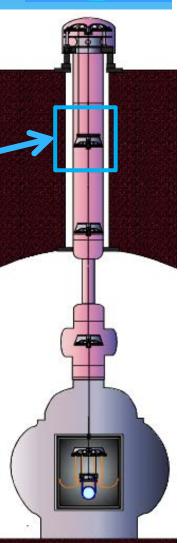


Torsion Mode Damping



* Eddy current damping for yaw modes



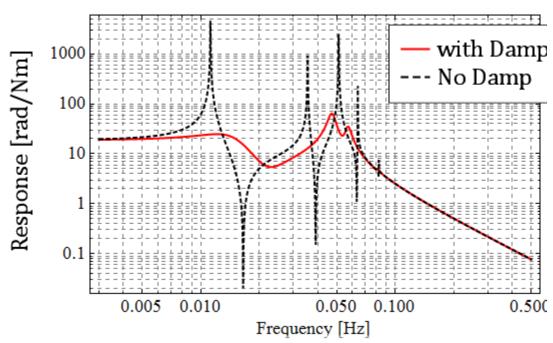




Torsion Mode Damping



* Torsion modes of the single wire suspensions would be sufficiently damped by the eddy current damper.



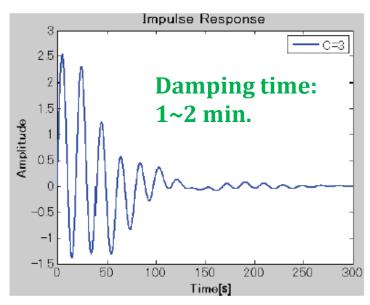


Figure 6.7: Impulse response to the external torque.

Mirror yaw angle response to applied torque



Effect on Angular Motion





