

Update on Suspensions for Enhanced and Advanced LIGO

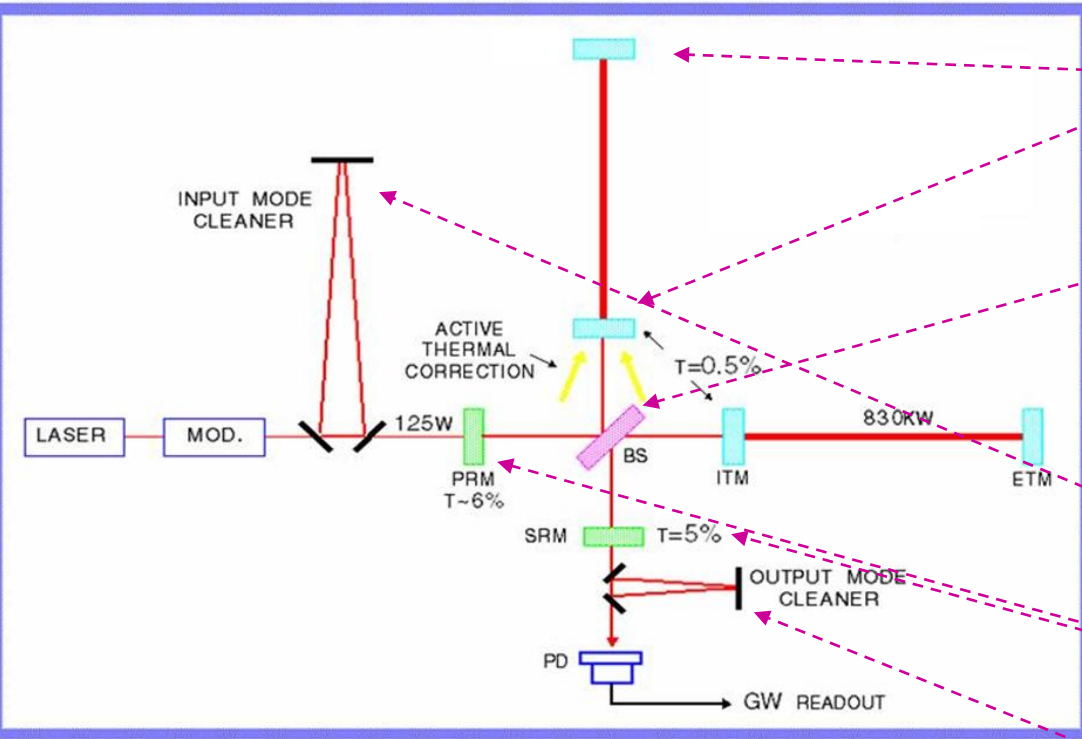
Norna A Robertson
LIGO-Caltech and University of Glasgow

For the Advanced LIGO SUS team

LSC/Virgo meeting, MIT
25th July 2007

G070482-00-R

- LIGO Caltech: R Abbott, H Armandula, A Campos (SURF), D Coyne, C Echols, J Heefner, B Kirsner, K Mailand, V Mandic, N Robertson (also at Glasgow), G Scarborough, S Waldman
- LIGO MIT: P Fritschel, R Mittleman, B Shapiro, N Smith
- LIGO LHO: B Bland
- LIGO LLO: J Romie
- University of Glasgow: M Barton, L Cunningham, A Cumming, A Heptonstall, J Hough, R Jones, I Martin, S Rowan, K Strain, C Torrie
- Rutherford Appleton Laboratory (RAL) : J Greenhalgh, T Hayler, J O'Dell, I Wilmut
- University of Birmingham: S Aston, R Cutler, D Lodhia
- University of Strathclyde: N Lockerbie

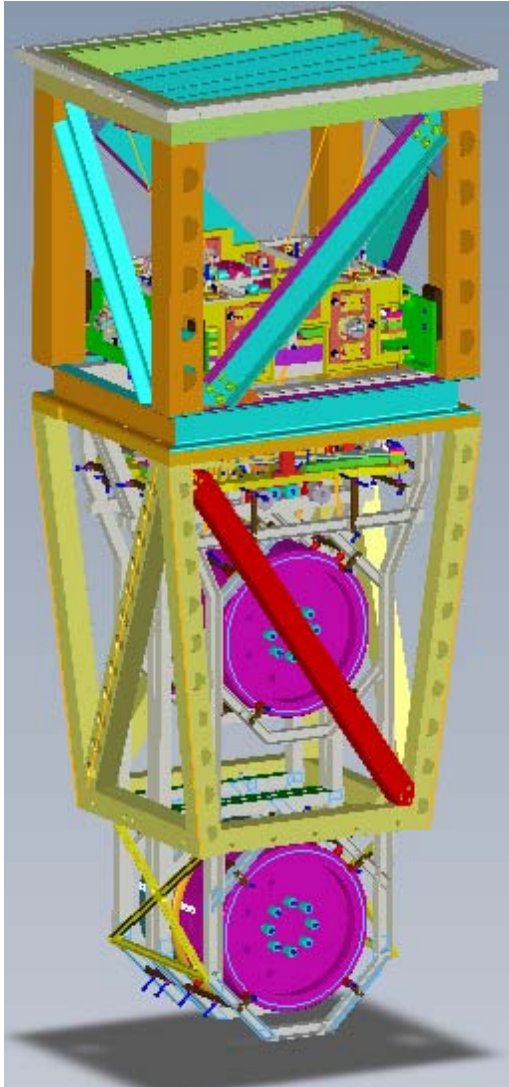


- Most sensitive optics in BSC chambers: UK deliverables
 - » Quadruple pendulum for Test Masses (ETM and ITM and reaction chains)
 - » Triple pendulum for Beamsplitter (BS) and Folding Mirror (FM)
- Other optics in HAM chambers: US deliverables
 - » Triple Pendulum for Input Modecleaner Mirrors (IMC)
 - » Triple pendulum for Power Recycling Mirror (PRM) and Signal Recycling Mirror (SRM)
 - » Double Pendulum for Output Modecleaner (OMC)

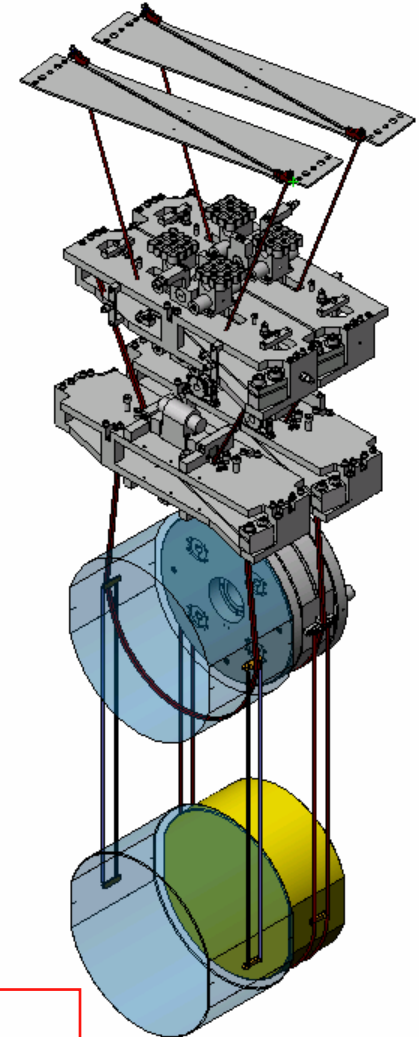
Advanced LIGO optical layout schematic

Test Mass

Quadruple Pendulum Suspension



- Key Design features
 - » Monolithic final stage: 40 kg fused silica mirror (34 cm diam x 20cm) on 4 fused silica ribbons (600mm x 1.1mm x 0.11 mm) for good thermal noise performance
 - » 4 stages for longitudinal seismic isolation plus 3 stages of blades for vertical isolation
 - » 6 degree of freedom damping (local control) at top mass for all low frequency modes (requires good mode coupling)
 - » Parallel reaction chain for quiet global control actuation: electrostatic (ESD) at test mass, electromagnetic at upper stages (hierarchical)

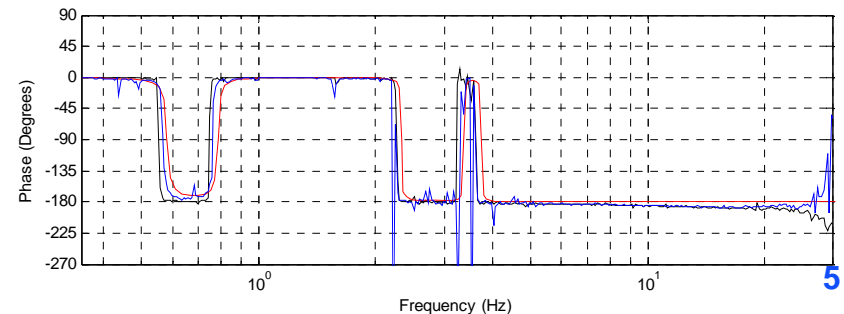
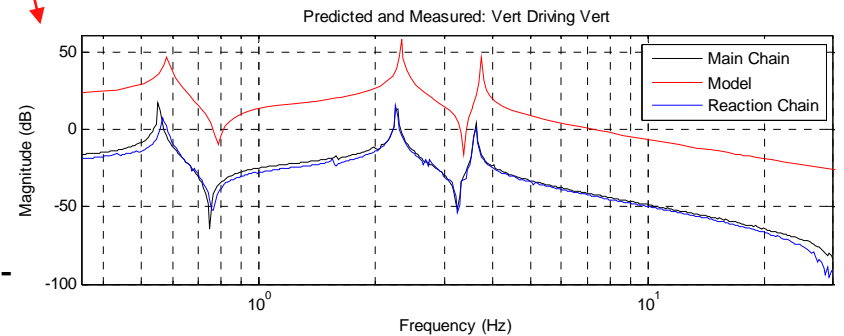


Major noise requirements

Suspension thermal noise $10^{-19} \text{ m}/\sqrt{\text{Hz}}$ @ 10 Hz
Residual seismic noise $10^{-19} \text{ m}/\sqrt{\text{Hz}}$ @ 10 Hz

- Current status

- » All-metal “controls” prototype assembled at Caltech and tested at LASTI. Assembly technique, mode frequencies, transfer functions, damping (active and eddy current) investigated. Compares well with revised Mathematica/MATLAB models
- » Lessons learnt re pitch and vertical modes: angled wires affect vertical frequencies and flexure lengths. Finite lateral blade compliance affects pitch.
- » Currently modal damping and cavity locking between quad and input modecleaner triple pendulum is being investigated (including use of electrostatic drive)
- » “Noise” prototype with full monolithic stage built at RAL and now under construction at LASTI (all-metal to start with)
- » Development of techniques for assembly of monolithic stage underway at Glasgow : CO₂ welding and pulling machine, profiler and proof/bounce tester, jigs for silicate bonding
- » OSEM and electronics builds ongoing (Birmingham and Strathclyde)



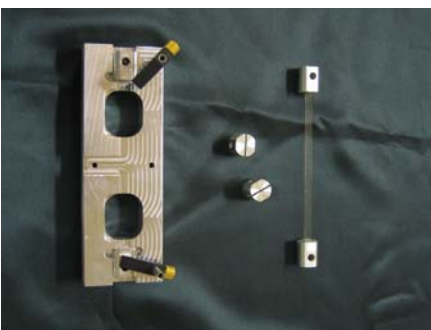
Development work on monolithic assembly techniques at Glasgow



CO₂ ribbon puller/welder



Prototype quad at Rutherford Appleton Lab



Preparing ribbon for pulling

Prism strength testing

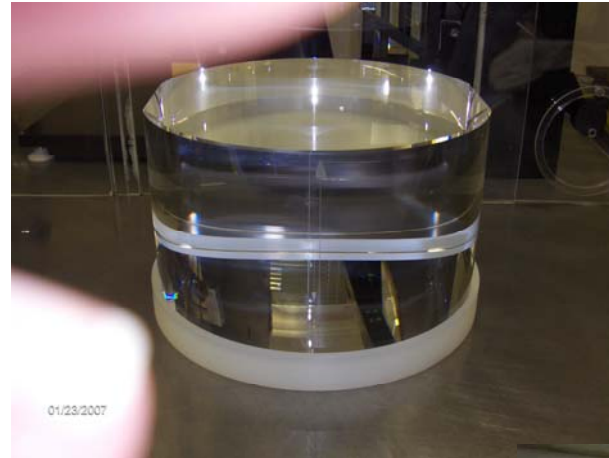


Ear bonding fixtures



- Future work

- » Full program of tests on noise prototype at LASTI over next ~9 months
 - mechanical fit, installation and alignment
 - interaction with seismic platform
 - mode frequencies, transfer functions and damping
 - thermal loading effects
 - assembly and reinstallation with monolithic stage
 - cavity tests incl. testing ESD
 - violin mode damping
- » OJEU (European Union) process for procurement of parts for final suspensions about to begin



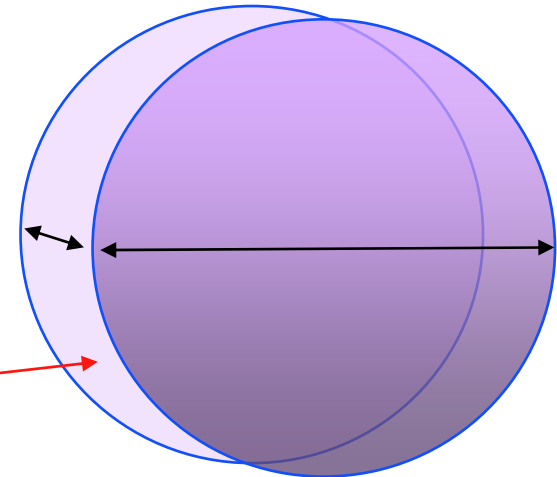
LASTI test mass

Prototype ring heater



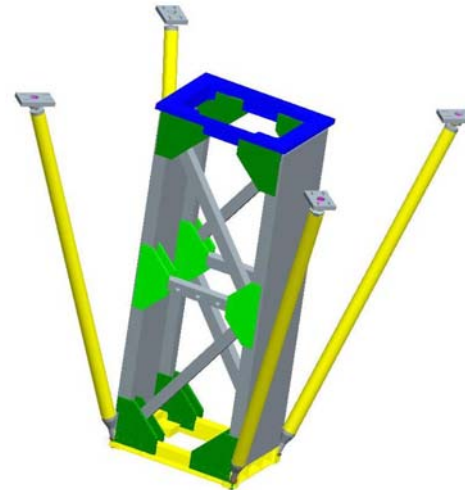
Reaction mass with 4 quadrant gold coating for electrostatic drive (ESD)

- Design features
 - » Triple pendulum with two stages of blades
 - » Silica mass 370 mm diam x 60 mm thick (~15 kg)
 - » Horizontal wedge
 - » Wire suspension (change of baseline)
- Design challenges
 - » Extreme mass aspect ratio (plate-like)
 - » Long thin support structure



Beamsplitter aspect ratio ~ 6 to 1

Prototype bolted structure at RAL

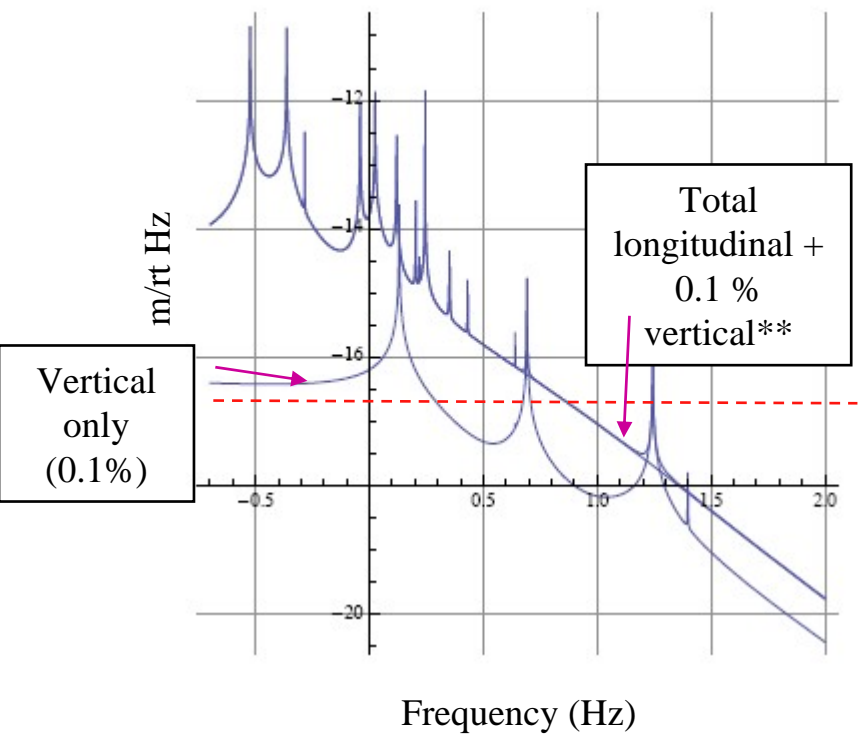


Struts to stiffen structure

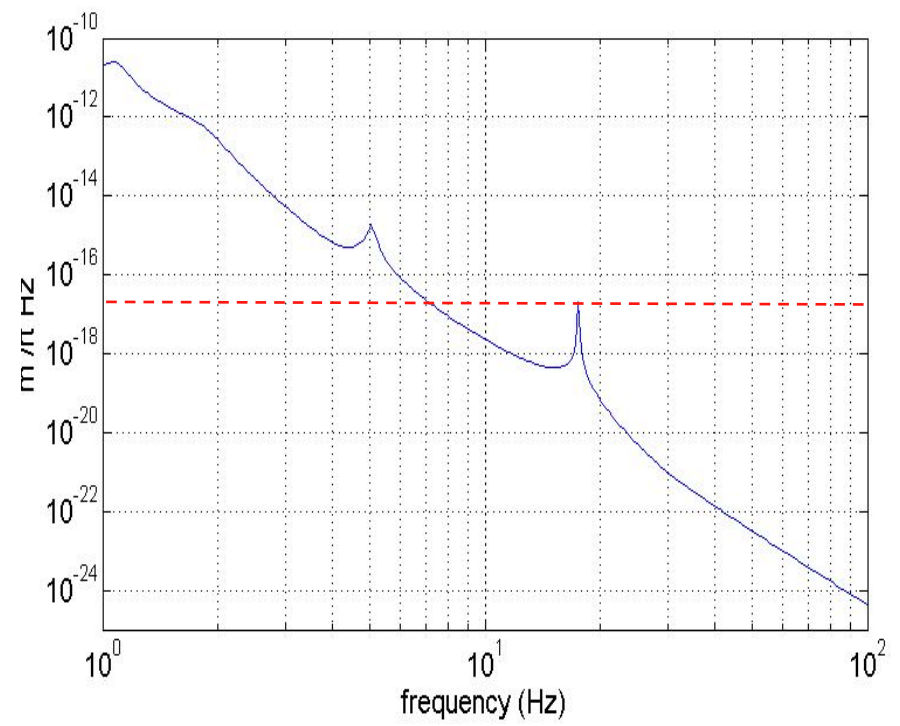


- Noise requirement: $2 \times 10^{-17} \text{ m}/\sqrt{\text{Hz}}$ at 10 Hz (-----)
- Noise estimates shown below, assuming 0.1% coupling from vertical

Thermal noise estimate

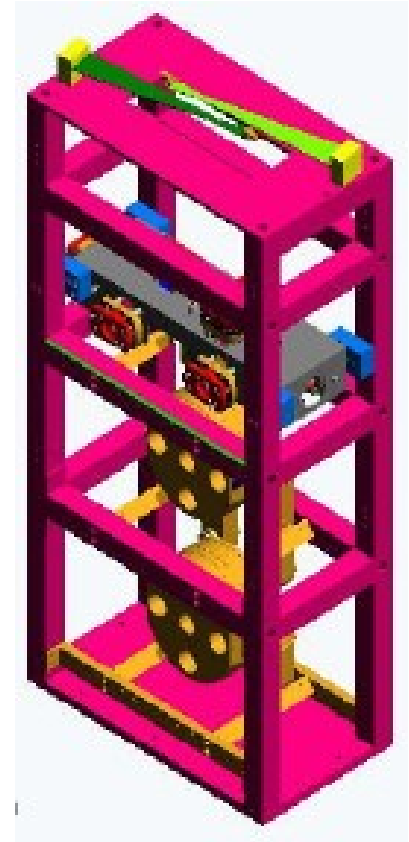


Seismic noise estimate*

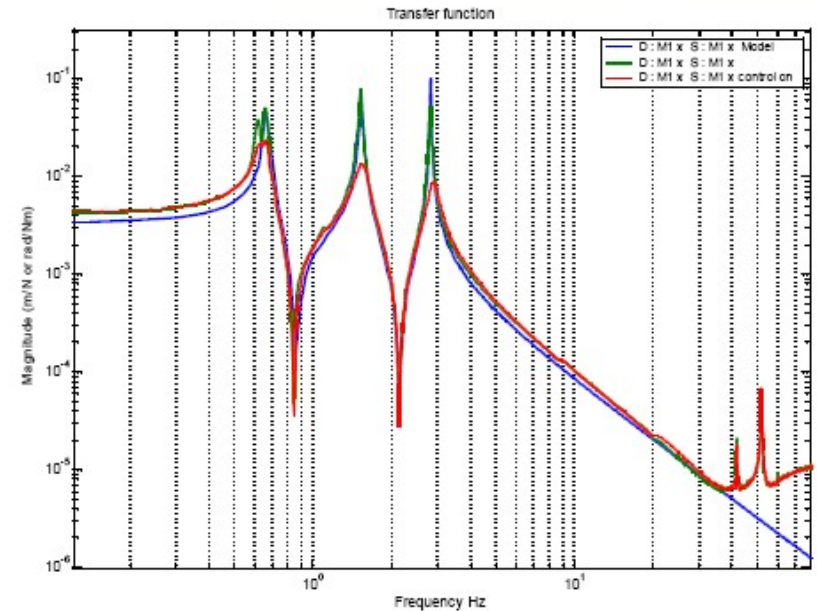


- Current status: at conceptual design stage

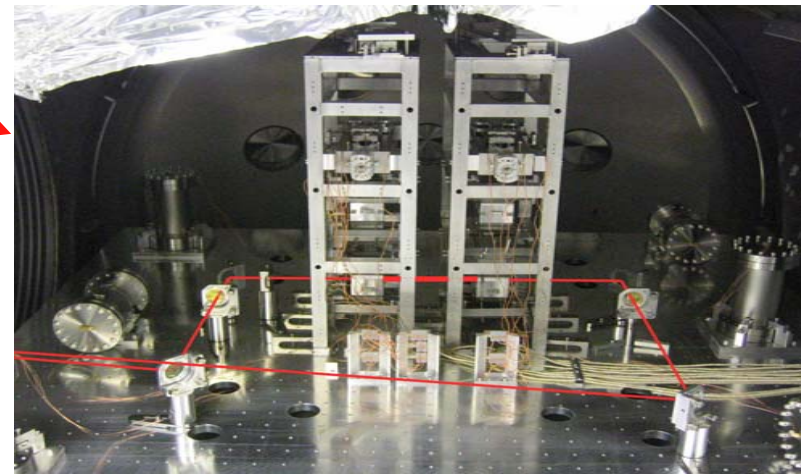
- Design features
 - » Triple pendulum with two stages of blades
 - » Silica mass: 150 mm diam. x 75 mm thick (~3 kg)
 - » Wire suspension
- Noise requirement: originally $3 \times 10^{-17} \text{ m}/\sqrt{\text{Hz}}$ at 10 Hz (relaxation of this requirement has since been proposed)
- Current status
 - » prototype fully characterised at LASTI
 - » currently being used for cavity tests with quad prototype



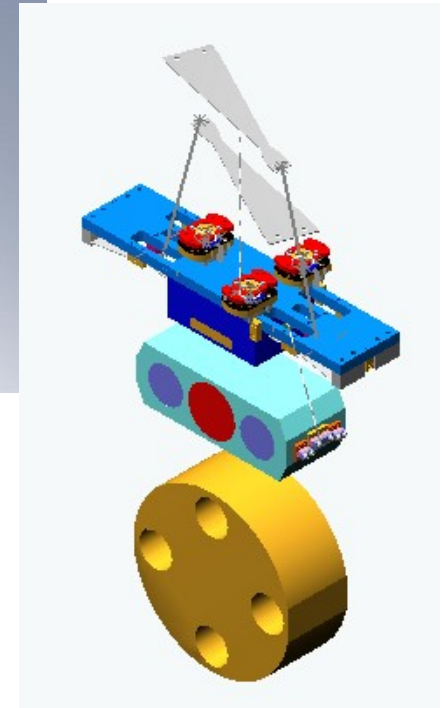
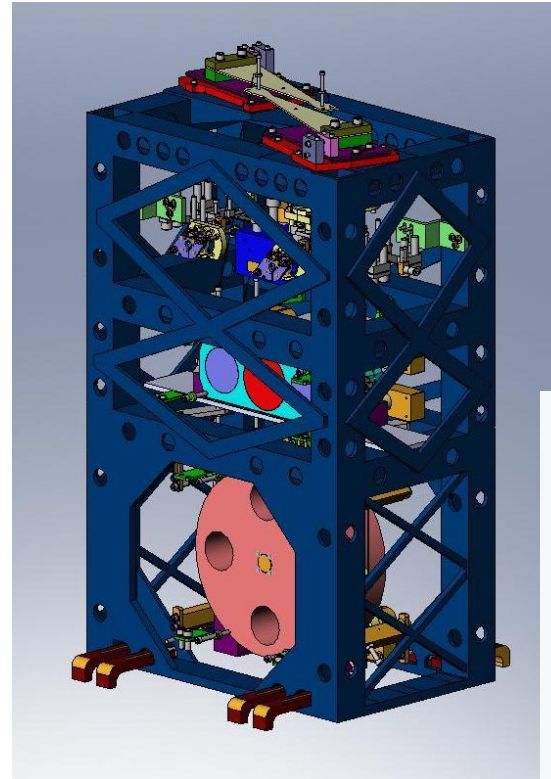
- Characterisation and comparison to MATLAB model
 - » Mode frequencies, transfer functions, active damping
- More recent work: test of independent modal control with a state estimator for damping - minimises sensor noise re-injection (L Ruet)
 - » relies on good model
 - » tested using second modecleaner triple pendulum as quiet reference
- Final design prototype testing & review: late 2008



Longitudinal transfer function at top mass
Blue: model, green: damping off, red: damping on



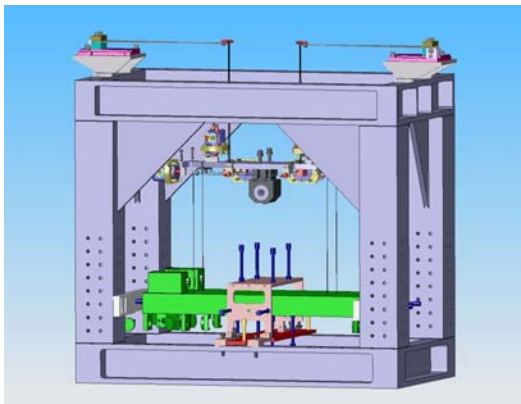
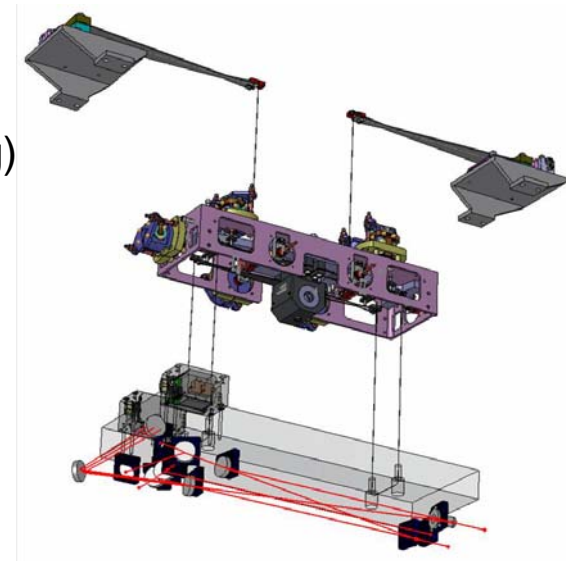
- Design features
 - » Triple pendulum with two stages of blades
 - » Silica mass: 265 mm diam. x 100 mm thick (~ 12 kg)
 - » Wire suspension
- Noise requirement
 - » $4 \times 10^{-16} \text{ m}/\sqrt{\text{Hz}}$ at 10 Hz
- Current status
 - » detailed design work now underway
 - » prototype assembly: early 2008



- Design Challenges
 - » Meeting height restriction with large mirror: middle mass “squashed”
 - » Meeting total mass budget (~ 120 kg) and 150 Hz lowest structural resonant frequency with steel structure
 - Stainless steel structure used to ease welding issues
 - » Access for assembly and adjustment
 - Mock-up of suspension underway.



- Design features
 - » Double pendulum with two stages of blades
 - » Silica optical bench 450 mm x 150 mm x 40 mm (~6 kg)
 - » Steel wires
- Requirements
 - » Double pendulum isolation and 6 DOF active damping
 - » Pendulum frequencies 0.8 to 2 Hz (guideline)
- Current status
 - » Prototype with metal bench under test
- First Installation
 - » In Enhanced LIGO, end of 2007

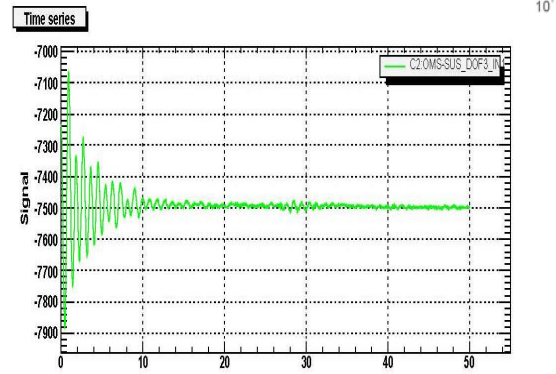
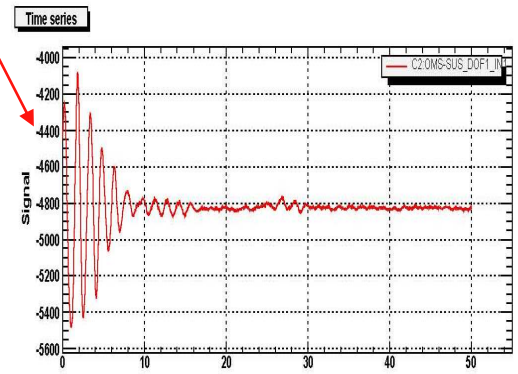
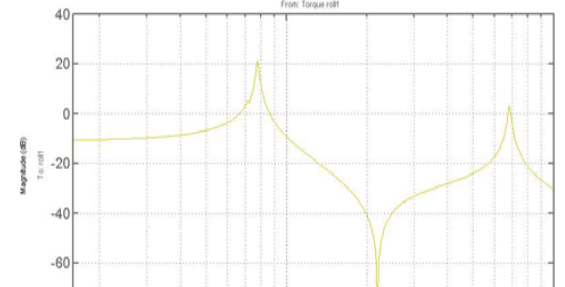
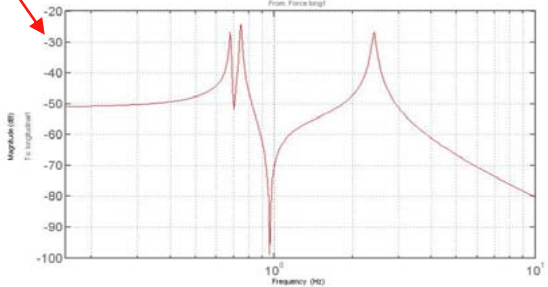
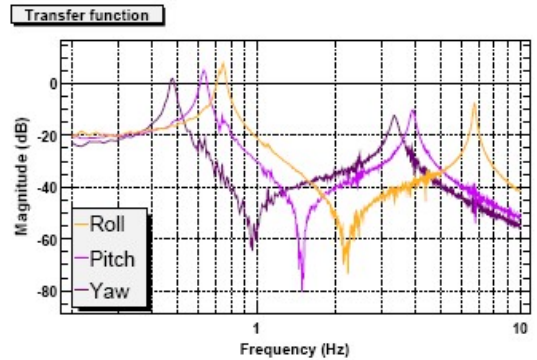
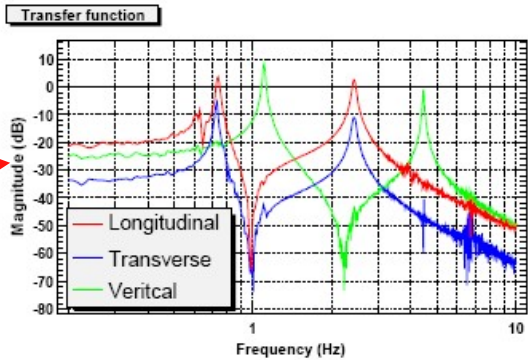
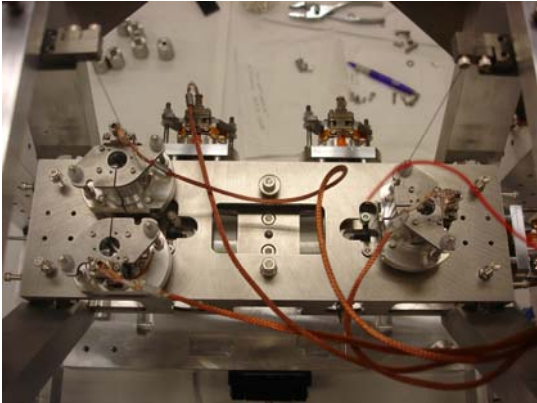


● Characterisation

- » Mode frequencies: from 0.5 Hz (lowest yaw mode) to 6.7 Hz (highest roll mode)
- » Transfer functions: measured and modelled (actuating and sensing at top mass)
- » Damping (long. and vert. shown)

● Other tests

- » Structural resonances
 - lowest ~148 Hz with ~4 kg non-suspended mass attached
- » OSEM design from UK
 - Works well



- All major suspension designs underway
- Common features:
 - » multiple pendulums with blades for vertical isolation
 - » active damping at top mass
 - designs based on GEO 600 design
- Distinctive features:
 - » Test mass quadruple pendulum: monolithic silica final stage with silicate bonding and silica ribbons, and use of reaction chain for global control
 - » All – different masses, sizes, requirements
- Status:
 - » prototypes of test mass quad, input modecleaner triple and output modecleaner double already tested
 - » others to follow
- Outlook:
 - » Promising!

